

Numerical Methods for Model Reduction of Large-Scale Dynamical Systems

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Thanks to

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- Enrique Quintana-Ortí, Gregorio Quintana-Ortí, Rafa Mayo, José Manuel Badía (Universidad Jaume I de Castellón).
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Problem

Given a physical problem with dynamics described by the **states** $x \in \mathbb{R}^n$, where n is the dimension of the **state space**.

Because of redundancies, complexity, etc., we want to describe the dynamics of the system using a reduced number of states.

This is the task of model reduction (also: dimension reduction, order reduction).



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*This is the task of **model reduction** (also: **dimension reduction**, **order reduction**).*



Example: Image Compression by Truncated SVD

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- A digital image with $n_x \times n_y$ pixels can be represented as matrix $X \in \mathbb{R}^{n_x \times n_y}$, where x_{ij} contains color information of pixel (i, j) .
- Memory: $4 \cdot n_x \cdot n_y$ bytes.

Theorem: (Schmidt-Mirsky/Eckart-Young)

Best rank- r approximation to $X \in \mathbb{R}^{n_x \times n_y}$ w.r.t. spectral norm:

$$\hat{X} = \sum_{k=1}^r \sigma_k u_k v_k^T,$$

where $X = U \Sigma V^T$ is the singular value decomposition (SVD) of X .
The approximation error is $\|X - \hat{X}\|_2 = \sigma_{k+1}$.

Idea for dimension reduction

Instead of X save $u_1, \dots, u_r, \sigma_1 v_1, \dots, \sigma_r v_r$.

\leadsto memory = $r \times (n_x + n_y)$ bytes.



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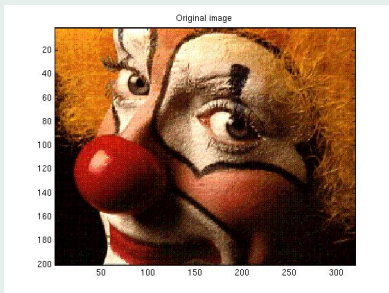
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Example: Clown





Example: Image Compression by Truncated SVD

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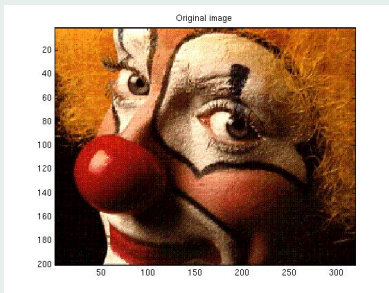
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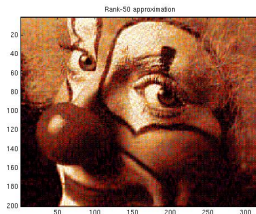
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Example: Clown



320×200 pixel
 $\rightsquigarrow \approx 256$ kb

■ rank $r = 50$, ≈ 104 kb





Example: Image Compression by Truncated SVD

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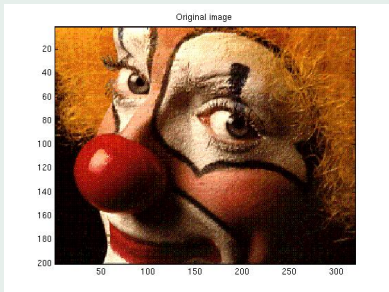
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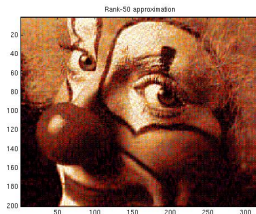
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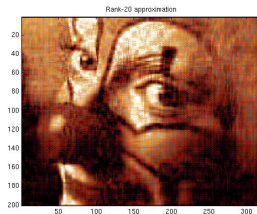
Example: Clown



■ rank $r = 50$, ≈ 104 kb



■ rank $r = 20$, ≈ 42 kb





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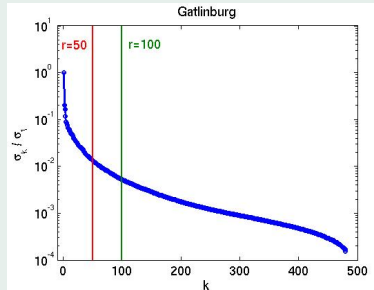
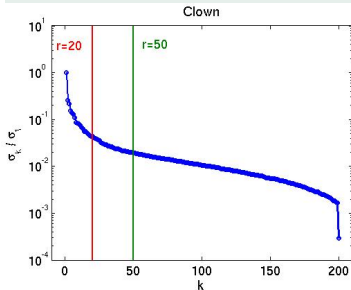
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Image data compression via SVD works, if the singular values decay (exponentially).

Singular Values of the Image Data Matrices





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Dynamical Systems

$$\Sigma : \begin{cases} \dot{x}(t) &= f(t, x(t), u(t)), \\ y(t) &= g(t, x(t), u(t)) \end{cases} \quad x(t_0) = x_0,$$

with

- **states** $x(t) \in \mathbb{R}^n$,
- **inputs** $u(t) \in \mathbb{R}^m$,
- **outputs** $y(t) \in \mathbb{R}^p$.





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Original System

$$\Sigma : \begin{cases} \dot{x}(t) = f(t, x(t), u(t)), \\ y(t) = g(t, x(t), u(t)). \end{cases}$$

- states $x(t) \in \mathbb{R}^n$,
- inputs $u(t) \in \mathbb{R}^m$,
- outputs $y(t) \in \mathbb{R}^p$.



Goal:

$$\|y - \hat{y}\| < \text{tolerance} \cdot \|u\| \text{ for all admissible input signals.}$$



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Reduced-Order System

$$\hat{\Sigma} : \begin{cases} \dot{\hat{x}}(t) = \hat{f}(t, \hat{x}(t), u(t)), \\ \hat{y}(t) = \hat{g}(t, \hat{x}(t), u(t)). \end{cases}$$

- states $\hat{x}(t) \in \mathbb{R}^r$, $r \ll n$
- inputs $u(t) \in \mathbb{R}^m$,
- outputs $\hat{y}(t) \in \mathbb{R}^p$.



Goal:

$\|y - \hat{y}\| < \text{tolerance} \cdot \|u\|$ for all admissible input signals.



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Linear, Time-Invariant (LTI) Systems

$$\begin{aligned} f(t, x, u) &= Ax + Bu, & A \in \mathbb{R}^{n \times n}, & B \in \mathbb{R}^{n \times m}, \\ g(t, x, u) &= Cx + Du, & C \in \mathbb{R}^{p \times n}, & D \in \mathbb{R}^{p \times m}. \end{aligned}$$

Laplace Transformation / Frequency Domain

Application of Laplace transformation ($x(t) \mapsto x(s)$, $\dot{x}(t) \mapsto sx(s)$) to linear system with $x(0) = 0$:

$$sx(s) = Ax(s) + Bu(s), \quad y(s) = Bx(s) + Du(s),$$

yields I/O-relation in frequency domain:

$$y(s) = \underbrace{\left(B(sI_n - A)^{-1}C + D \right)}_{=: G(s)} u(s)$$

G is the transfer function of Σ .



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Application of **Laplace transformation** ($x(t) \mapsto x(s)$, $\dot{x}(t) \mapsto sx(s)$) to linear system with $x(0) = 0$:

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Approximate the dynamical system

$$\begin{aligned}\dot{x} &= Ax + Bu, & A \in \mathbb{R}^{n \times n}, & B \in \mathbb{R}^{n \times m}, \\ y &= Cx + Du, & C \in \mathbb{R}^{p \times n}, & D \in \mathbb{R}^{p \times m}.\end{aligned}$$

by reduced-order system

$$\begin{aligned}\dot{\hat{x}} &= \hat{A}\hat{x} + \hat{B}u, & \hat{A} \in \mathbb{R}^{r \times r}, & \hat{B} \in \mathbb{R}^{r \times m}, \\ \hat{y} &= \hat{C}\hat{x} + \hat{D}u, & \hat{C} \in \mathbb{R}^{p \times r}, & \hat{D} \in \mathbb{R}^{p \times m}.\end{aligned}$$

of order $r \ll n$, such that

$$\|y - \hat{y}\| = \|Gu - \hat{G}u\| \leq \|G - \hat{G}\| \|u\| < \text{tolerance} \cdot \|u\|.$$

\implies Approximation problem: $\min_{\text{order}(\hat{G}) \leq r} \|G - \hat{G}\|$.



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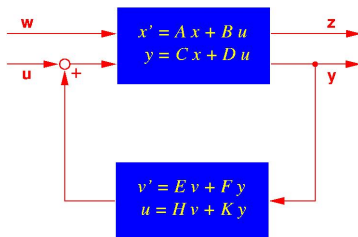
Feedback Controllers

A feedback controller (**dynamic compensator**) is a linear system of order N , where

- input = output of plant,
- output = input of plant.

Modern (LQG-/ \mathcal{H}_2 -/ \mathcal{H}_∞ -) control design: $N \geq n$

\Rightarrow reduce order of original system.





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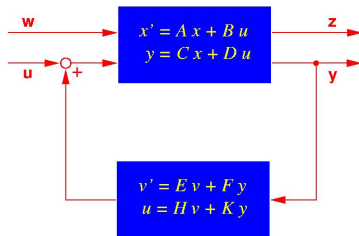
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- **Progressive miniaturization: Moore's Law** states that the number of on-chip transistors doubles each 12 (now: 18) months.
- Verification of VLSI/ULSI chip design requires high number of simulations for different input signals.
- Increase in packing density requires modeling of interconnect to ensure that thermic/electro-magnetic effects do not disturb signal transmission.
- Linear systems in micro electronics occur through modified nodal analysis (MNA) for RLC networks, e.g., when
 - decoupling large linear subcircuits,
 - modeling transmission lines,
 - modeling pin packages in VLSI chips,
 - modeling circuit elements described by Maxwell's equation using partial element equivalent circuits (PEEC).



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Example for Miniaturization

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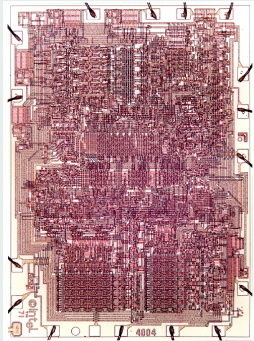
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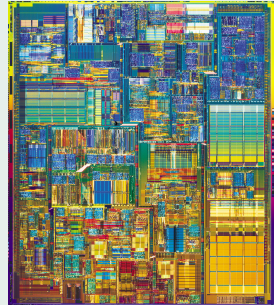
References

Intel 4004 (1971)



- 1 layer, 10μ technology,
- 2,300 transistors,
- 64 kHz clock speed.

Intel Pentium IV (2001)



- 7 layers, 0.18μ technology,
- 42,000,000 transistors,
- 2 GHz clock speed,
- 2km of interconnect.



MEMS/Microsystems

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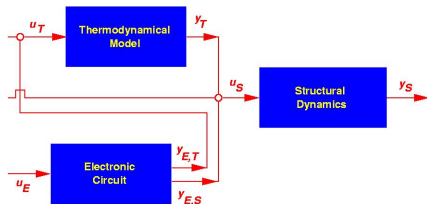
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Typical problem in MEMS simulation:
coupling of different models (thermic, structural, electric, electro-magnetic) during simulation.

Problems and Challenges:

- Reduce simulation times by replacing sub-systems with their reduced-order models.
- Stability properties of coupled system may deteriorate through model reduction even when stable sub-systems are replaced by stable reduced-order models.
- Multi-scale phenomena.





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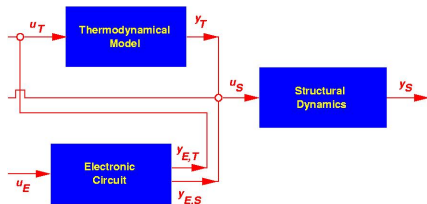
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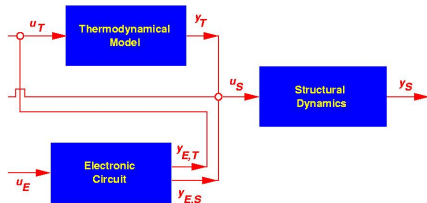
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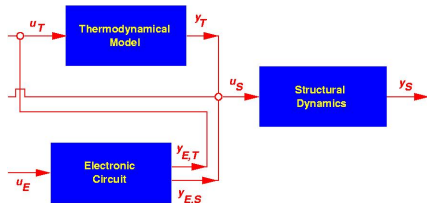
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- **Automatic generation of compact models.**
- Satisfy desired error tolerance for all admissible input signals, i.e., want

$$\|y - \hat{y}\| < \text{tolerance} \cdot \|u\| \quad \forall u \in L_2(\mathbb{R}, \mathbb{R}^m).$$

⇒ Need computable error bound/estimate!

- Preserve physical properties:

• preserve passivity of SISO
• preserve passivity (control of SISO)
• preserve (dissipativity of SISO)
• preserve (dissipativity of SISO)



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\Rightarrow Need computable error bound/estimate!

- **Preserve physical properties:**
 - stability (poles of G in \mathbb{C}^-),
 - minimum phase (zeroes of G in \mathbb{C}^-),
 - passivity (“system does not generate energy”).



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- 1 Modal Truncation
- 2 Guyan-Reduction/Substructuring
- 3 Padé-Approximation and Krylov Subspace Methods
- 4 Balanced Truncation
- 5 many more. . .



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- 4 Balanced Truncation
- 5 many more...

Joint feature of many methods: **Galerkin** or **Petrov-Galerkin-type projection** of state-space onto low-dimensional subspace \mathcal{V} along \mathcal{W} : assume $x \approx VW^T x =: \tilde{x}$, where

$$\text{range}(V) = \mathcal{V}, \quad \text{range}(W) = \mathcal{W}, \quad W^T V = I_r.$$

Then, with $\hat{x} = W^T x$, we obtain $x \approx V\hat{x}$ and

$$\|x - \tilde{x}\| = \|x - V\hat{x}\|.$$



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Idea:

Project state-space onto A -invariant subspace \mathcal{V} , where

$$\mathcal{V} = \text{span}(v_1, \dots, v_r),$$

v_k = eigenvectors corresp. to “dominant” **modes** \equiv **eigenvalues** of A .



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$$\mathcal{V} = \text{span}(v_1, \dots, v_r),$$

v_k = eigenvectors corresp. to “dominant” modes \equiv eigenvalues of A .

Properties:

- Simple computation for large-scale systems, using, e.g., Krylov subspace methods (Lanczos, Arnoldi), Jacobi-Davidson method.
- Error bound:

$$\|G - \hat{G}\|_{\infty} \leq \text{cond}_2(T) \|C_2\|_2 \|B_2\|_2 \frac{1}{\min_{\lambda \in \Lambda(A_2)} |\text{Re}(\lambda)|},$$

where $T^{-1}AT = \text{diag}(A_1, A_2)$.



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$$\mathcal{V} = \text{span}(v_1, \dots, v_r),$$

v_k = eigenvectors corresp. to “dominant” modes \equiv eigenvalues of A .

Difficulties:

- Eigenvalues contain only limited system information.
- Dominance measures are difficult to compute.
(LITZ 1979: use Jordan canonical form; otherwise merely heuristic criteria.)
- Error bound not computable for really large-scale problems.



Guyan Reduction (Static Condensation)

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Partition states in inner and outer (**master**) nodes; eliminate inner nodes in stationary system.



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References

Partition states in inner and outer (master) nodes; eliminate inner nodes in stationary system.

Properties:

- + Simple calculation for large-scale systems with definite A -matrix, using, e.g., CG algorithm.



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- + Natural approach in connection with **domain decomposition methods**.



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- ± In ANSYS implemented for dimension reduction.



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- ± Hierarchical application (**substructuring**) using the modal basis (**Craig-Bampton method**) yields efficient methods for applications in structural mechanics.



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- ± In ANSYS implemented for dimension reduction.
- ± Hierarchical application (**substructuring**) using the modal basis (**Craig-Bampton method**) yields efficient methods for applications in structural mechanics.
- Non-static behavior of the system is ignored.



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Idea:

- Consider

$$E\dot{x} = Ax + Bu, \quad y = Cx$$

with rational transfer function $G(s) = C(sE - A)^{-1}B$.



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Idea:

- Consider

$$E\dot{x} = Ax + Bu, \quad y = Cx$$

with rational transfer function $G(s) = C(sE - A)^{-1}B$.

- For $s_0 \notin \Lambda(A, E)$:

$$G(s) = m_0 + m_1(s - s_0) + m_2(s - s_0)^2 + \dots$$



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$$G(s) = m_0 + m_1(s - s_0) + m_2(s - s_0)^2 + \dots$$

- As reduced-order model use **rth Padé approximate** \hat{G} to G :

$$G(s) = \hat{G}(s) + \mathcal{O}((s - s_0)^{2r}),$$

i.e., $m_j = \hat{m}_j$ for $j = 0, \dots, 2r - 1$

\rightsquigarrow **moment matching** if $s_0 < \infty$,

\rightsquigarrow **partial realization** if $s_0 = \infty$.



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Padé-via-Lanczos Method (PVL)

- Moments need not be computed explicitly; moment matching is equivalent to **projecting** state-space **onto**

$$\mathcal{V} = \text{span}(\tilde{B}, \tilde{A}\tilde{B}, \dots, \tilde{A}^{r-1}\tilde{B}) = \mathcal{K}(\tilde{A}, \tilde{B}, r)$$

(where $\tilde{A} = (s_0 E - A)^{-1} E$, $\tilde{B} = (s_0 E - A)^{-1} B$) along

$$\mathcal{W} = \text{span}(C^H, \tilde{A}^H C^H, \dots, (\tilde{A}^H)^{r-1} C^H) = \mathcal{K}(\tilde{A}^H, C^H, r).$$



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- Computation via unsymmetric Lanczos method, yields system matrices of reduced-order model as by-product.



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- Computation via unsymmetric Lanczos method, yields system matrices of reduced-order model as by-product.
- PVL applies w/o changes for singular E if $s_0 \notin \Lambda(A, E)$:
 - for $s_0 \neq \infty$: GALLIVAN/GRIMME/VAN DOOREN 1994,
FREUND/FELDMANN 1996, GRIMME 1997
 - for $s_0 = \infty$: B./SOKOLOV 2005



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Padé-via-Lanczos Method (PVL)

Difficulties:

- No computable error estimates/bounds for $\|y - \hat{y}\|_2$.



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Padé-via-Lanczos Method (PVL)

Difficulties:

- No computable error estimates/bounds for $\|y - \hat{y}\|_2$.
- Mostly heuristic criteria for choice of expansion points.
Optimal choice for second-order systems with proportional/Rayleigh damping (BEATTIE/GUGERCIN 2005).



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- No computable error estimates/bounds for $\|y - \hat{y}\|_2$.
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Optimal choice for second-order systems with proportional/Rayleigh damping (BEATTIE/GUGERCIN 2005).
- Good approximation quality only locally.
- Preservation of physical properties only in very special cases; usually requires post processing which (partially) destroys moment matching properties.



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Idea:

- A system Σ , realized by (A, B, C, D) , is called **balanced**, if solutions P, Q of the **Lyapunov equations**

$$AP + PA^T + BB^T = 0, \quad A^T Q + QA + C^T C = 0,$$

satisfy: $P = Q = \text{diag}(\sigma_1, \dots, \sigma_n)$ with $\sigma_1 \geq \sigma_2 \geq \dots \geq \sigma_n > 0$.



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- $\{\sigma_1, \dots, \sigma_n\}$ are the **Hankel singular values (HSVs)** of Σ .



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satisfy: $P = Q = \text{diag}(\sigma_1, \dots, \sigma_n)$ with $\sigma_1 \geq \sigma_2 \geq \dots \geq \sigma_n > 0$.

- $\{\sigma_1, \dots, \sigma_n\}$ are the Hankel singular values (HSVs) of Σ .
- Compute balanced realization of the system via **state-space transformation**

$$\begin{aligned} \mathcal{T} : (A, B, C, D) &\mapsto (TAT^{-1}, TB, T^{-1}C, D) \\ &= \left(\begin{bmatrix} \textcolor{red}{A}_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix}, \begin{bmatrix} \textcolor{red}{B}_1 \\ B_2 \end{bmatrix}, \begin{bmatrix} \textcolor{red}{C}_1 & C_2 \end{bmatrix}, \textcolor{red}{D} \right) \end{aligned}$$



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- Truncation $\rightsquigarrow (\hat{A}, \hat{B}, \hat{C}, \hat{D}) = (A_{11}, B_1, C_1, D)$.



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Motivation:

HSV are **system invariants**: they are preserved under \mathcal{T} and determine the energy transfer given by the Hankel map

$$\mathcal{H} : L_2(-\infty, 0) \mapsto L_2(0, \infty) : u_- \mapsto y_+.$$



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$$\mathcal{H} : L_2(-\infty, 0) \mapsto L_2(0, \infty) : u_- \mapsto y_+.$$

In balanced coordinates ... **energy transfer from u_- to y_+** :

$$E := \sup_{\substack{u \in L_2(-\infty, 0] \\ x(0) = x_0}} \frac{\int_0^\infty y(t)^T y(t) dt}{\int_{-\infty}^0 u(t)^T u(t) dt} = \frac{1}{\|x_0\|_2} \sum_{j=1}^n \sigma_j^2 x_{0,j}^2$$



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⇒ **Truncate states corresponding to “small” HSVs**

⇒ **complete analogy to best approximation via SVD!**



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Properties:

- Reduced-order model is stable with HSVs $\sigma_1, \dots, \sigma_r$.



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Properties:

- Reduced-order model is stable with HSVs $\sigma_1, \dots, \sigma_r$.
- Adaptive choice of r via computable error bound:

$$\|y - \hat{y}\|_2 \leq \left(2 \sum_{k=r+1}^n \sigma_k \right) \|u\|_2.$$



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Properties:

- Reduced-order model is stable with HSVs $\sigma_1, \dots, \sigma_r$.

- Adaptive choice of r via computable error bound:

$$\|y - \hat{y}\|_2 \leq \left(2 \sum_{k=r+1}^n \sigma_k\right) \|u\|_2.$$

- Several related methods by variation of Gramians for
 - closed-loop model reduction (LQG balancing),
 - minimum-phase preservation (balanced stochastic truncation),
 - passivity preservation (positive-real balanced truncation).



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Properties:

General misunderstanding: complexity $\mathcal{O}(n^3)$ – true for several implementations! (e.g., MATLAB, SLICOT).



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Properties:

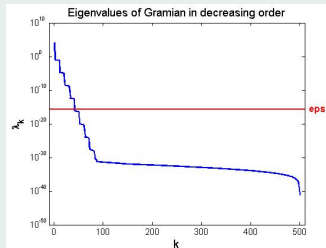
General misunderstanding: complexity $\mathcal{O}(n^3)$ – true for several implementations! (e.g., MATLAB, SLICOT).

New algorithmic ideas from numerical linear algebra:

- Instead of Gramians P, Q compute $\hat{S}, \hat{R} \in \mathbb{R}^{n \times k}$, $k \ll n$, such that

$$P \approx \hat{S}\hat{S}^T, \quad Q \approx \hat{R}\hat{R}^T.$$

- Compute \hat{S}, \hat{R} with problem-specific Lyapunov solvers of “low” complexity directly.





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New algorithmic ideas from numerical linear algebra:

Parallelization:

- Efficient parallel algorithms based on matrix sign function.
- **Complexity $\mathcal{O}(n^3/q)$** on q -processor machine.
- Software library **PLICMR** with **WebComputing interface**.

(B./QUINTANA-ORTÍ/QUINTANA-ORTÍ since 1999)



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(B./QUINTANA-ORTÍ/QUINTANA-ORTÍ since 1999)

Formatted Arithmetic:

For special problems from PDE control use implementation based on hierarchical matrices and matrix sign function method (BAUR/B.), **complexity $\mathcal{O}(n \log^2(n) r^2)$** .



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Properties:

General misunderstanding: complexity $\mathcal{O}(n^3)$ – true for several implementations! (e.g., MATLAB, SLICOT).

New algorithmic ideas from numerical linear algebra:

Sparse Balanced Truncation:

- Sparse implementation using sparse Lyapunov solver (ADI+MUMPS/SuperLU).
- **Complexity $\mathcal{O}(n(k^2 + r^2))$.**
- Software:
 - + MATLAB toolbox **LYAPACK** (PENZL 1999),
 - + Software library **SPARED** with **WebComputing interface**.
(BADÍA/B./QUINTANA-ORTÍ/QUINTANA-ORTÍ since 2003)



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Why is Balanced Truncation Superior?

Consider the approximation problem:

project x onto r -dim. subspace $\mathcal{V} \subset \mathbb{R}^n$ such that $\|x - V\hat{x}\| = \min!$



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- Modal truncation chooses from the $\binom{n}{r}$ many A -invariant subspaces.



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Why is Balanced Truncation Superior?

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- Modal truncation chooses from the $\binom{n}{r}$ many A -invariant subspaces.
- PVL chooses exactly **one** subspace (the Krylov subspace $\mathcal{K}(\tilde{A}, \tilde{B})$).



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Why is Balanced Truncation Superior?

Consider the approximation problem:

project x onto r -dim. subspace $\mathcal{V} \subset \mathbb{R}^n$ such that $\|x - V\hat{x}\| = \min!$

- Modal truncation chooses from the $\binom{n}{r}$ many A -invariant subspaces.
- PVL chooses exactly one subspace (the Krylov subspace $\mathcal{K}(\tilde{A}, \tilde{B})$).
- Balanced truncation can choose \mathcal{V} from the complete **Grassman manifold**

$$\mathcal{G}(n, r) = \{\mathcal{V} \subset \mathbb{R}^n : \dim \mathcal{V} = r\}.$$



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$$\mathcal{G}(n, r) = \{\mathcal{V} \subset \mathbb{R}^n : \dim \mathcal{V} = r\}.$$

Consequence: BT often needs the least states for a prescribed accuracy/yields the best accuracy for a prescribed number of states.



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Why is Balanced Truncation **Not Always** Superior?

Consider the approximation problem:

project x onto r -dim. subspace $\mathcal{V} \subset \mathbb{R}^n$ such that $\|x - V\hat{x}\| = \min!$

- Modal truncation in practice
 - corrects larger error by static condensation and
 - makes an informed choice of modes based on a-priori knowledge about input signals.



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- Modal truncation in practice
 - corrects larger error by static condensation and
 - makes an informed choice of modes based on a-priori knowledge about input signals.
- PVL pre-selects a “good” subspace by picking the expansion points close to assumed operating frequency.
- Balanced truncation aims at global minimization and thereby sometimes neglects local features.



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Optimal Control: Cooling of Steel Profiles

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- Mathematical model: boundary control for linearized 2D heat equation.

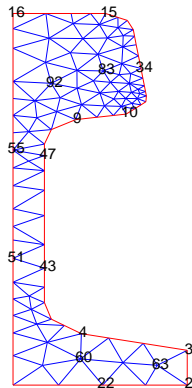
$$c \cdot \rho \frac{\partial}{\partial t} x = \lambda \Delta x, \quad \xi \in \Omega$$

$$\lambda \frac{\partial}{\partial n} x = u_k - x, \quad \xi \in \Gamma_k, \quad 1 \leq k \leq 7,$$

$$\frac{\partial}{\partial n} x = 0, \quad \xi \in \Gamma_7.$$

$$\implies m = 7, p = 6.$$

- FEM Discretization, different models for initial mesh ($n = 371$),
1, 2, 3, 4 steps of mesh refinement \implies
 $n = 1357, 5177, 20209, 79841$.



Source: Physical model: courtesy of Mannesmann/Demag.

Math. model: TRÖLTZSCH/UNGER 1999/2001, PENZL 1999, SAAK 2003.



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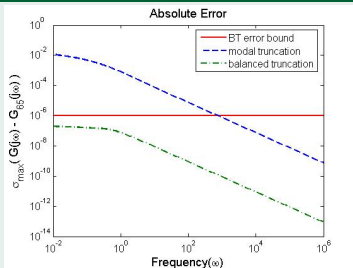
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$n = 1357$, Absolute Error



- BT model computed with sign function method,
- MT w/o static condensation, same order as BT model.



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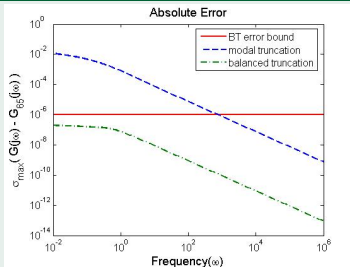
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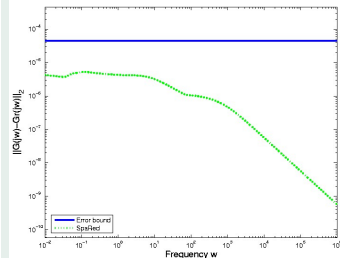
References

$n = 1357$, Absolute Error



- BT model computed with sign function method,
- MT w/o static condensation, same order as BT model.

$n = 79841$, Absolute error



- BT model computed using SpaRed,
- computation time: **8 min.**



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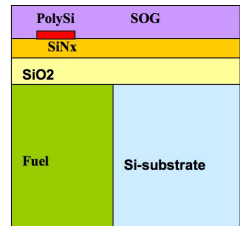
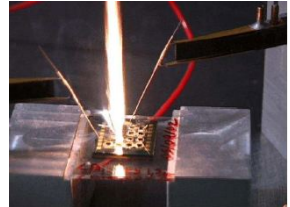
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- Co-integration of solid fuel with silicon micromachined system.
- Goal: Ignition of solid fuel cells by electric impulse.
- Application: nano satellites.
- Thermo-dynamical model, ignition via heating an electric resistance by applying voltage source.
- Design problem: reach ignition temperature of fuel cell w/o firing neighbouring cells.
- Spatial FEM discretization of thermo-dynamical model \rightsquigarrow linear system, $m = 1$, $p = 7$.



Source: The Oberwolfach Benchmark Collection <http://www.imtek.de/simulation/benchmark>

Courtesy of C. Rossi, LAAS-CNRS/EU project "Micropyros".



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MEMS: Microthruster

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- axial-symmetric 2D model
- FEM discretisation using linear elements $\rightsquigarrow n = 4,257, m = 1, p = 7$.
- Reduced model computed using SPARED and Arnoldi(A, B).
- Order of reduced model: $r = 30$ ($r = 120$ for Arnoldi).



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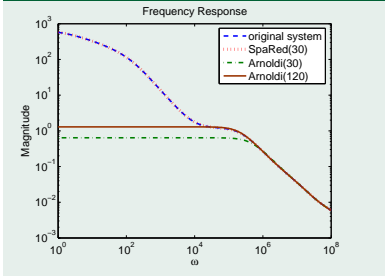
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Frequency Response Analysis





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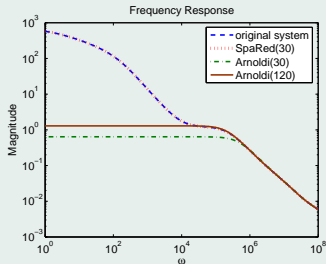
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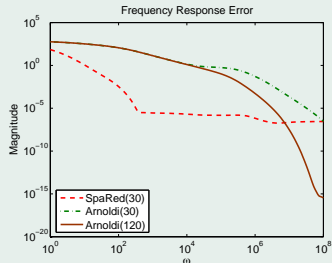
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Frequency Response Analysis



Absolute error





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MEMS: Microgyroscope (Butterfly Gyro)

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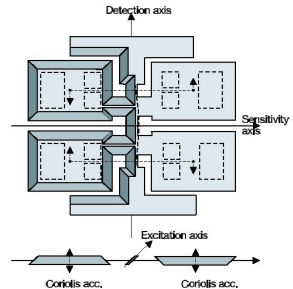
Future Work

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- By applying AC voltage to electrodes, wings are forced to vibrate in anti-phase in wafer plane.
- Coriolis forces induce motion of wings out of wafer plane yielding sensor data.

- Vibrating micro-mechanical gyroscope for inertial navigation.
- Rotational position sensor.



Source: The Oberwolfach Benchmark Collection <http://www.imtek.de/simulation/benchmark>

Courtesy of D. Billger (Imeko Institute, Göteborg), Saab Bofors Dynamics AB.



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- FEM discretization of structure dynamical model using quadratic tetrahedral elements (ANSYS-SOLID187)
 $\rightsquigarrow n = 34,722, m = 1, p = 12.$
- Reduced model computed using SPARED, $r = 30.$



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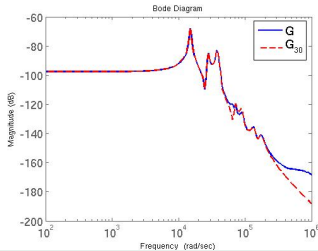
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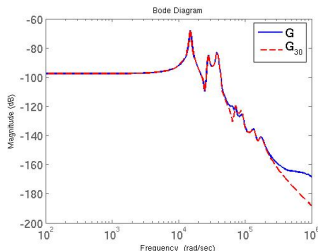
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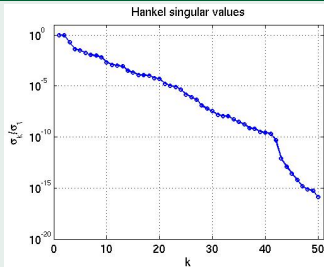
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Frequency Repsonse Analysis



Hankel Singular Values





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Parametric Models

$$\dot{x} = A(p)x + B(p)u, \quad y = C(p)x + D(p)u,$$

where $p \in \mathbb{R}^s$ are free parameters which should be preserved in the reduced-order model.



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Parametric Models

$$\dot{x} = A(p)x + B(p)u, \quad y = C(p)x + D(p)u,$$

where $p \in \mathbb{R}^s$ are free parameters which should be preserved in the reduced-order model.

- Frequently: B, C, D parameter independent,

$$A(p) = A_0 + p_1 A_1 + \dots + p_s A_s.$$

\Rightarrow (Modified) linear model reduction methods applicable.

- **Multipoint expansion** combined with Padé-type approx. possible.



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Parametric Models

$$\dot{x} = A(p)x + B(p)u, \quad y = C(p)x + D(p)u,$$

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- Frequently: B, C, D parameter independent,

$$A(p) = A_0 + p_1 A_1 + \dots + p_s A_s.$$

⇒ (Modified) linear model reduction methods applicable.

- Multipoint expansion combined with Padé-type approx. possible.
- **New idea:** BT for reference parameters combined with interpolation yields parametric reduced-order models.



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Parametric Models

Nonlinear Systems

- Linear projection

$$x \approx V\hat{x}, \quad \dot{\hat{x}} = W^T f(V\hat{x}, u)$$

is in general not model reduction!



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Parametric Models

Nonlinear Systems

- Linear projection

$$x \approx V\hat{x}, \quad \dot{\hat{x}} = W^T f(V\hat{x}, u)$$

is in general not model reduction!

- Need specific methods
 - POD + balanced truncation \rightsquigarrow empirical Gramians (LALL/MARSDEN/GLAVASKI 1999/2002),
 - Approximate inertial manifold method (\sim static condensation for nonlinear systems).



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Parametric Models

Nonlinear Systems

- Linear projection

$$x \approx V\hat{x}, \quad \dot{\hat{x}} = W^T f(V\hat{x}, u)$$

is in general not model reduction!

- Exploit structure of nonlinearities, e.g., in optimal control of linear PDEs with nonlinear BCs \rightsquigarrow
 - bilinear control systems $\dot{x} = Ax + \sum_j N_j x u_j + Bu$,
 - formal linear systems (cf. FÖLLINGER 1982)

$$\dot{x} = Ax + N g(Hx) + Bu = Ax + \begin{bmatrix} B & N \end{bmatrix} \begin{bmatrix} u \\ g(z) \end{bmatrix},$$

where $z := Hx \in \mathbb{R}^\ell$, $\ell \ll n$.



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Thanks for getting up early!